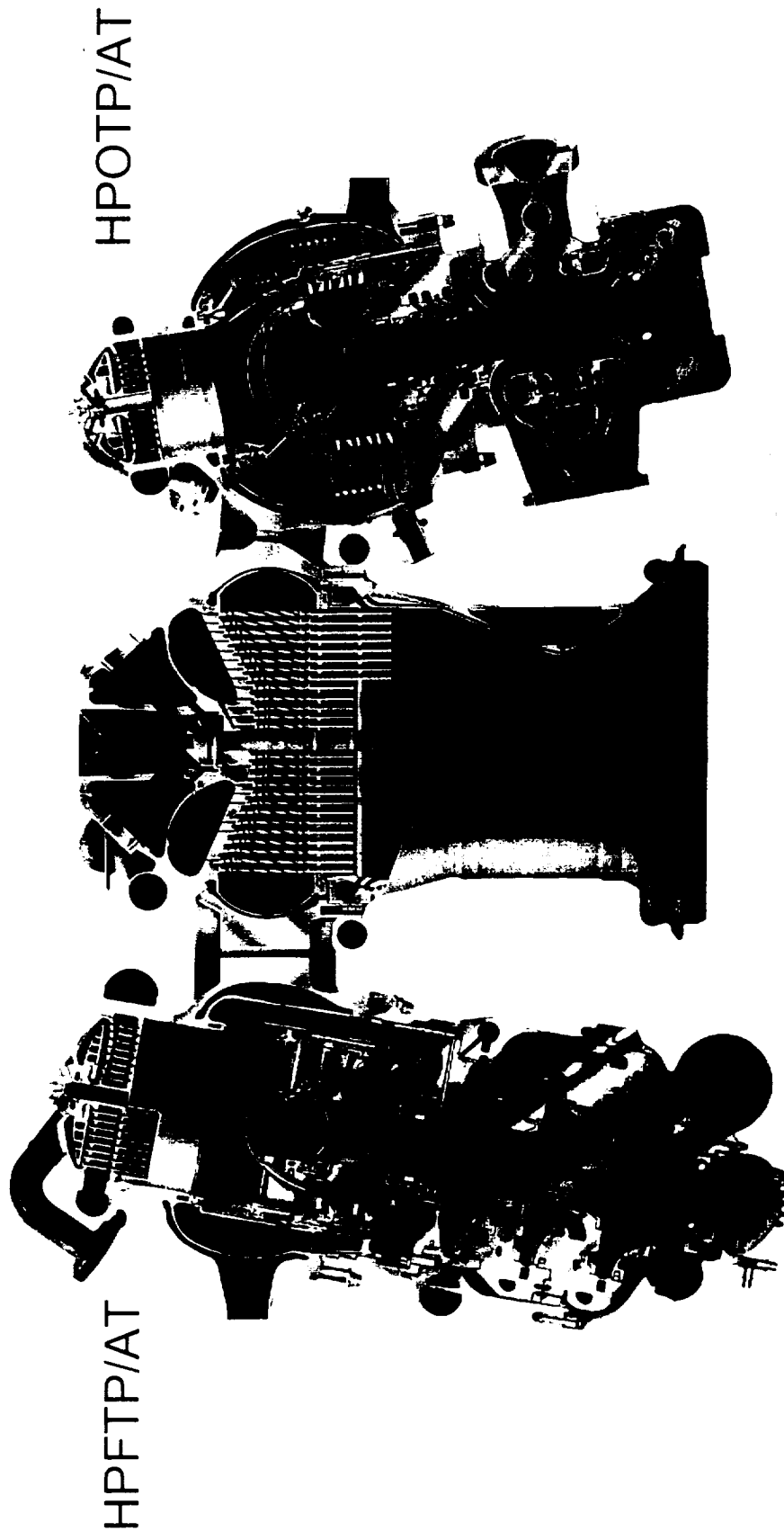




Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

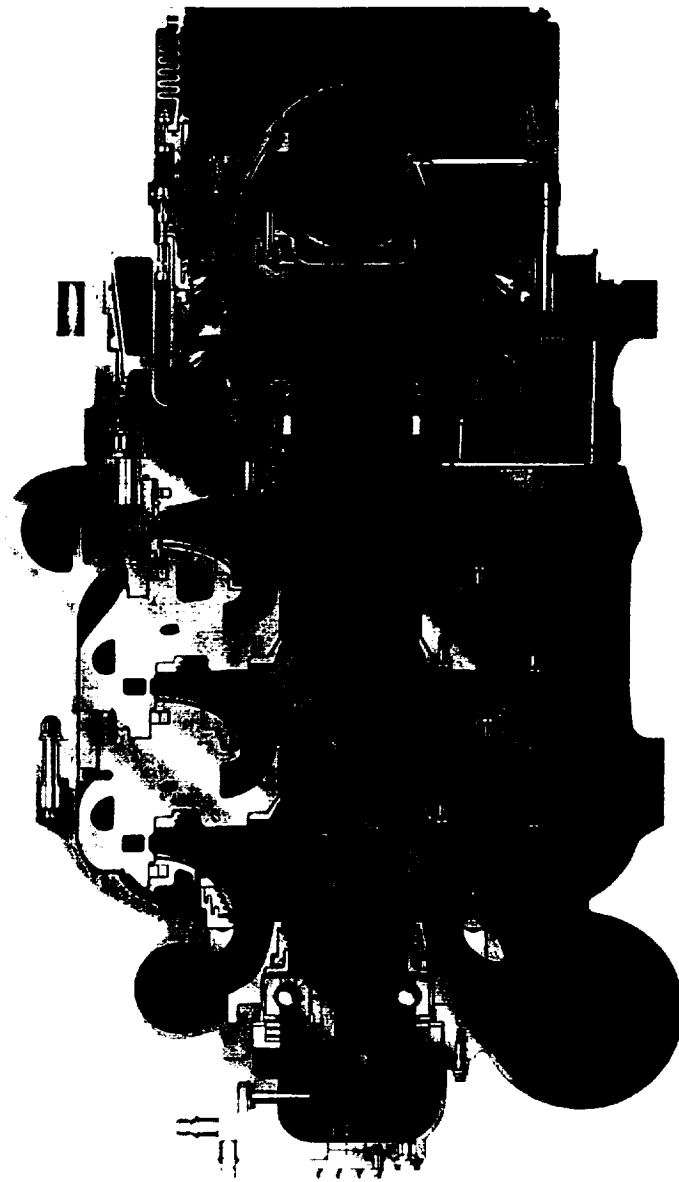
SSME POWERHEAD





Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

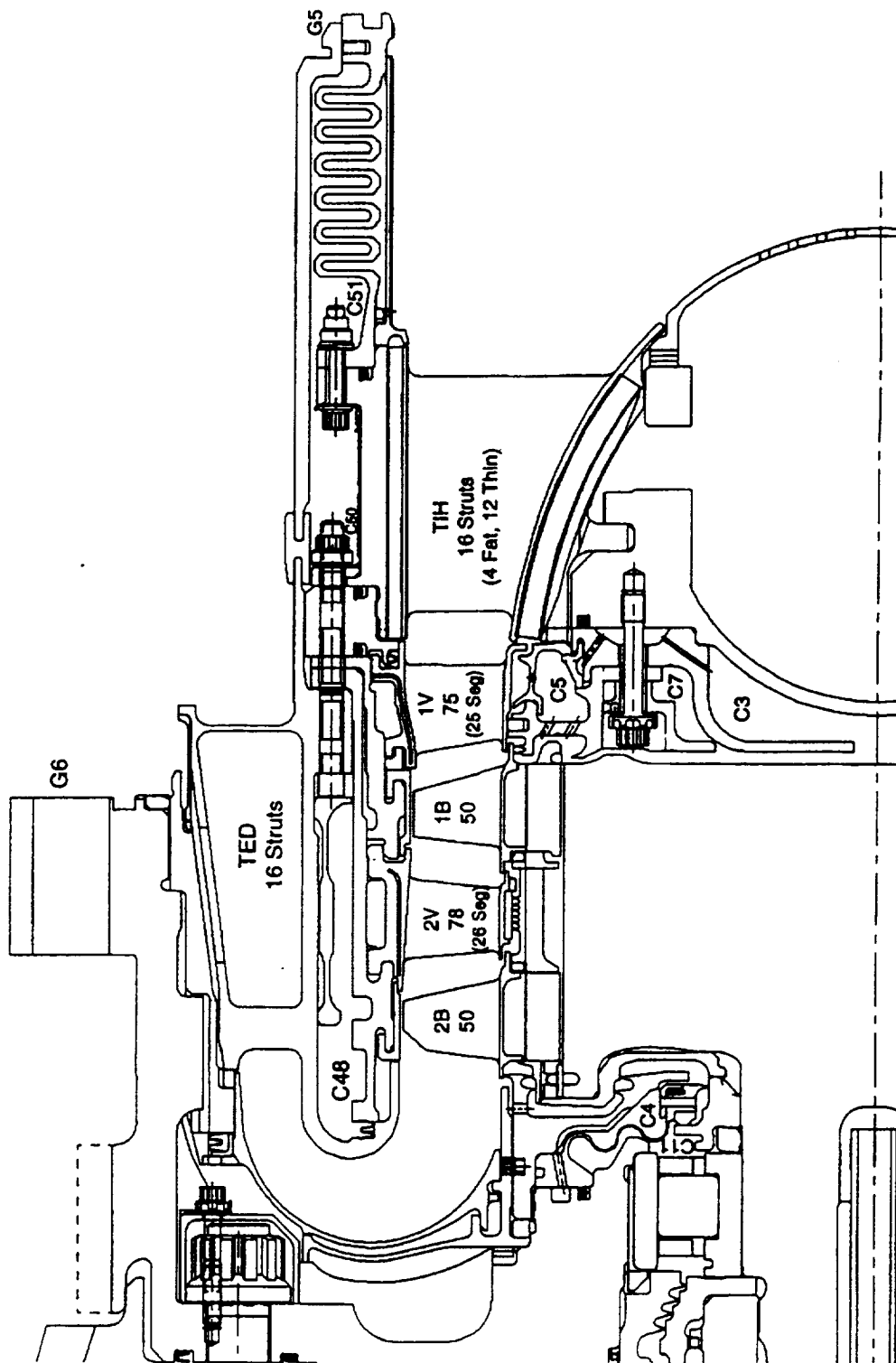
HPFTP/AT



FC0126400



Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades





Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

Agenda

- Single Crystal Material
- Uniaxial LCF Specimen Data
- Development Blade Failure Analysis



Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

Single Crystal Materials

- Anisotropic Material

- Nickel based (face centered cubic structure)
- Properties vary with orientation
- Contains preferential easy slip systems

- No Grain Boundaries

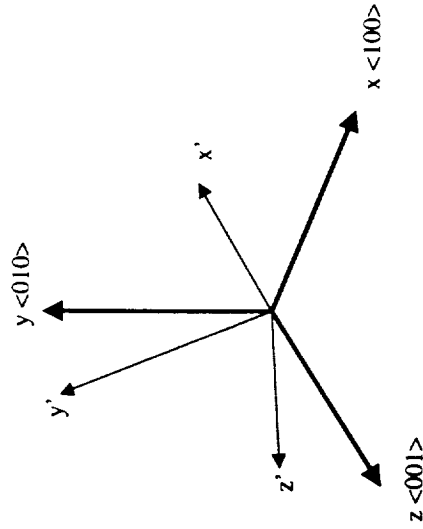
- Higher strength at elevated temperature
- Greater creep resistance at elevated temperature

- Possible to Control Crystal Orientation Within A Structure

- P&W controls the primary material axis to within 15 degrees of the stacking axis
- The secondary axis is uncontrolled



Coordinate Transformations for Orthotropic Material



Hooke's Law

$$\begin{aligned}\{\sigma\} &= [D] \{\epsilon\} \\ \{\sigma'\} &= [D'] \{\epsilon'\} \\ [D'] &= [Q]^T [D] [Q]\end{aligned}$$

$[Q] = (6 \times 6)$ matrix of direction cosines

Single crystal material can be thought of as a 3D composite. The material properties are orthotropic and vary with orientation. Orthotropic materials with cubic symmetry have 3 independent material properties: E, ν , and G.

$$\{\sigma'\} = [Q'] \{\sigma\}$$

$$\{\sigma\} = [Q']^{-1} \{\sigma'\} = [Q] \{\sigma'\}$$

$$\{\epsilon'\} = [Q_\epsilon'] \{\epsilon\}$$

$$\{\epsilon\} = [a_\epsilon] \{\sigma\}$$

$$a_{11} = \frac{1}{E_{xx}}, a_{44} = \frac{1}{G_{yz}}, a_{12} = -\frac{\nu_{yx}}{E_{xx}} = -\frac{\nu_{xy}}{E_{yy}}$$

$$\{\epsilon\} = [a_\epsilon] \{\sigma\}$$

$$\{\epsilon'\} = [a_\epsilon'] \{\sigma'\}$$

$$[a'_{ij}] = [Q]^T [a_{ij}] [Q] = \sum_{m=1}^6 \sum_{n=1}^6 a_{mn} Q_{mi} Q_{nj} \quad (i, j = 1, 2, \dots, 6)$$

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

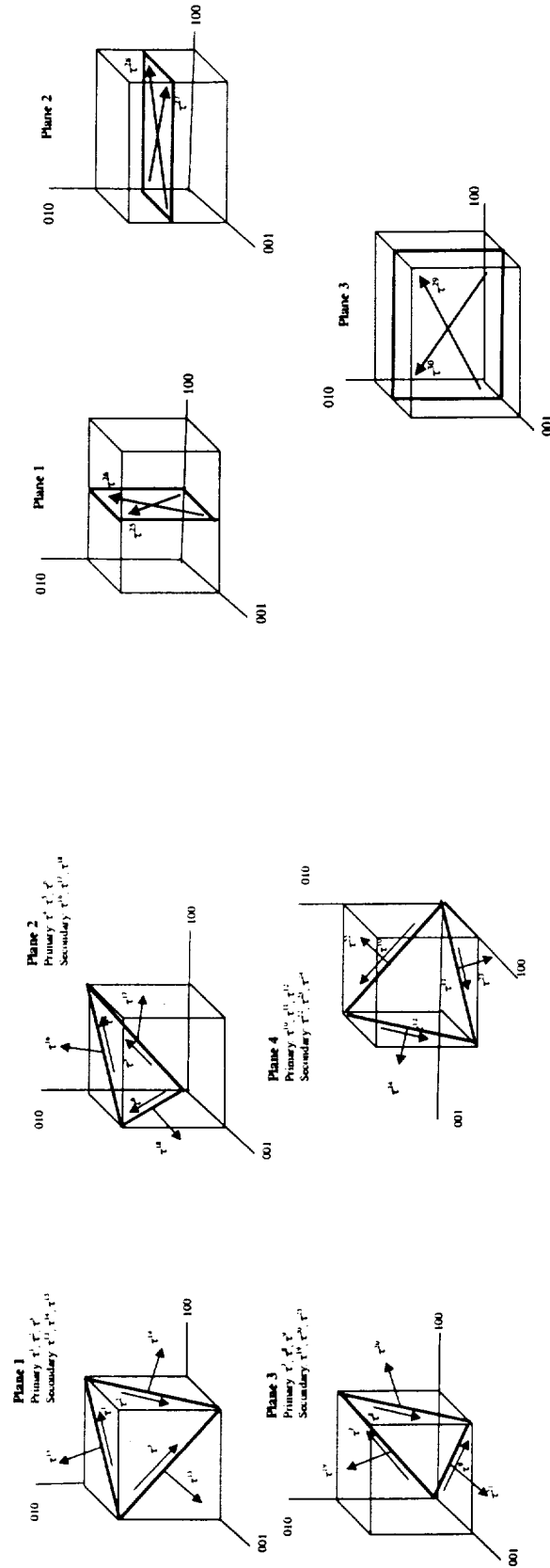


Fig. 3.1 Primary (close-pack) and secondary (non-close-pack) slip directions on the octahedral planes for a FCC crystal ⁴

Fig. 3.2 Cube slip planes and slip directions for a FCC crystal ⁴

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



$$\begin{Bmatrix} \tau^1 \\ \tau^2 \\ \tau^3 \\ \tau^4 \\ \tau^5 \\ \tau^6 \\ \tau^7 \\ \tau^8 \\ \tau^9 \\ \tau^{10} \\ \tau^{11} \\ \tau^{12} \end{Bmatrix} = \frac{1}{\sqrt{6}} \begin{bmatrix} 1 & 0 & -1 & 1 & 0 & -1 \\ 0 & -1 & 1 & -1 & 1 & 0 \\ 1 & -1 & 0 & 0 & 1 & -1 \\ -1 & 0 & 1 & 1 & 0 & -1 \\ -1 & 1 & 0 & 0 & -1 & 1 \\ 0 & 1 & -1 & -1 & 1 & 0 \\ 1 & -1 & 0 & 0 & -1 & 1 \\ 0 & 1 & -1 & 1 & 0 & -1 \\ 1 & 0 & -1 & -1 & 0 & -1 \\ 0 & -1 & 1 & -1 & -1 & 0 \\ -1 & 0 & 1 & -1 & 0 & -1 \\ -1 & 1 & 0 & 0 & 1 & -1 \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{Bmatrix}$$

$$= \frac{1}{3\sqrt{2}} \begin{bmatrix} -1 & 2 & -1 & 1 & -2 & 1 \\ 2 & -1 & -1 & 1 & 1 & -2 \\ -1 & -1 & 2 & -2 & 1 & 1 \\ -1 & 2 & -1 & -1 & -2 & -1 \\ -1 & -1 & 2 & 2 & 1 & -1 \\ 2 & -1 & -1 & 2 & -1 & 1 \\ -1 & -1 & -1 & 2 & -1 & 1 \\ 2 & -1 & -1 & -1 & -2 & -1 \\ -1 & 2 & -1 & -1 & 2 & 1 \\ 2 & -1 & -1 & 1 & -1 & 2 \\ -1 & 2 & -1 & 1 & 2 & -1 \\ -1 & -1 & 2 & -2 & -1 & -1 \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{Bmatrix}$$

$$\begin{Bmatrix} \gamma^1 \\ \gamma^2 \\ \gamma^3 \\ \gamma^4 \\ \gamma^5 \\ \gamma^6 \\ \gamma^7 \\ \gamma^8 \\ \gamma^9 \\ \gamma^{10} \\ \gamma^{11} \\ \gamma^{12} \end{Bmatrix} = \frac{2}{\sqrt{6}} \begin{bmatrix} 1 & 0 & -1 & 1 & 0 & -1 \\ 0 & -1 & 1 & -1 & 1 & 0 \\ 1 & -1 & 0 & 0 & 1 & -1 \\ -1 & 0 & 1 & 1 & 0 & -1 \\ -1 & 1 & 0 & 0 & -1 & 1 \\ 0 & 1 & -1 & -1 & 1 & 0 \\ 1 & -1 & 0 & 0 & -1 & 1 \\ 0 & 1 & -1 & 1 & 0 & -1 \\ 1 & 0 & -1 & -1 & 0 & -1 \\ 0 & -1 & 1 & -1 & -1 & 0 \\ -1 & 0 & 1 & -1 & 0 & -1 \\ -1 & 1 & 0 & 0 & 1 & -1 \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{xy} \\ \epsilon_{yz} \\ \epsilon_{zx} \end{Bmatrix}$$

$$= \frac{2}{3\sqrt{2}} \begin{bmatrix} -1 & 2 & -1 & 1 & -2 & 1 \\ 2 & -1 & -1 & 1 & 1 & -2 \\ -1 & -1 & 2 & -2 & 1 & 1 \\ -1 & 2 & -1 & -1 & -2 & -1 \\ -1 & -1 & 2 & 2 & 1 & -1 \\ 2 & -1 & -1 & 2 & -1 & 1 \\ -1 & -1 & -1 & 2 & -1 & 1 \\ 2 & -1 & -1 & -1 & -2 & -1 \\ -1 & 2 & -1 & -1 & 2 & 1 \\ 2 & -1 & -1 & 1 & -1 & 2 \\ -1 & 2 & -1 & 1 & 2 & -1 \\ -1 & -1 & 2 & -2 & -1 & -1 \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{xy} \\ \epsilon_{yz} \\ \epsilon_{zx} \end{Bmatrix}$$

$$\begin{Bmatrix} \tau^{13} \\ \tau^{14} \\ \tau^{15} \\ \tau^{16} \\ \tau^{17} \\ \tau^{18} \end{Bmatrix} = \frac{1}{\sqrt{2}} \begin{bmatrix} 0 & 0 & 0 & 1 & 1 & 0 \\ 0 & 0 & 0 & 1 & -1 & 0 \\ 0 & 0 & 0 & 1 & 0 & 1 \\ 0 & 0 & 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 1 & 1 & 1 \\ 0 & 0 & 0 & -1 & 1 & 1 \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{Bmatrix}$$

$$\begin{Bmatrix} \gamma^{13} \\ \gamma^{14} \\ \gamma^{15} \\ \gamma^{16} \\ \gamma^{17} \\ \gamma^{18} \end{Bmatrix} = \frac{2}{\sqrt{2}} \begin{bmatrix} 0 & 0 & 0 & 1 & 1 & 0 \\ 0 & 0 & 0 & 1 & -1 & 0 \\ 0 & 0 & 0 & 1 & 0 & 1 \\ 0 & 0 & 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 1 & 1 & 1 \\ 0 & 0 & 0 & -1 & 1 & 1 \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{xy} \\ \epsilon_{yz} \\ \epsilon_{zx} \end{Bmatrix}$$

Resolved shear stresses from the
material coordinate system stress
tensor

Resolved engineering shear strains
from the material coordinate system
strain tensor

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

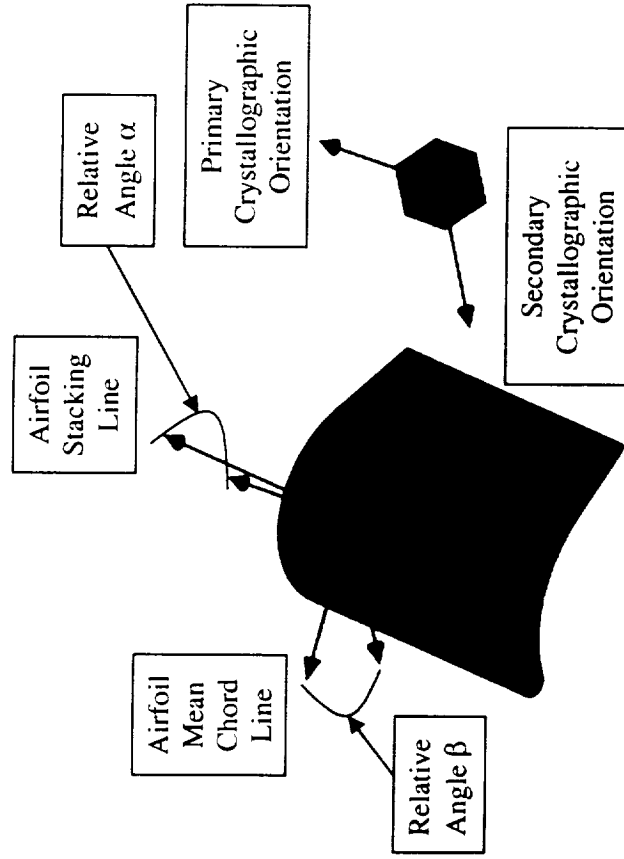


Fig. 2.4 Convention for Defining Crystal Orientation in Turbine Blades²

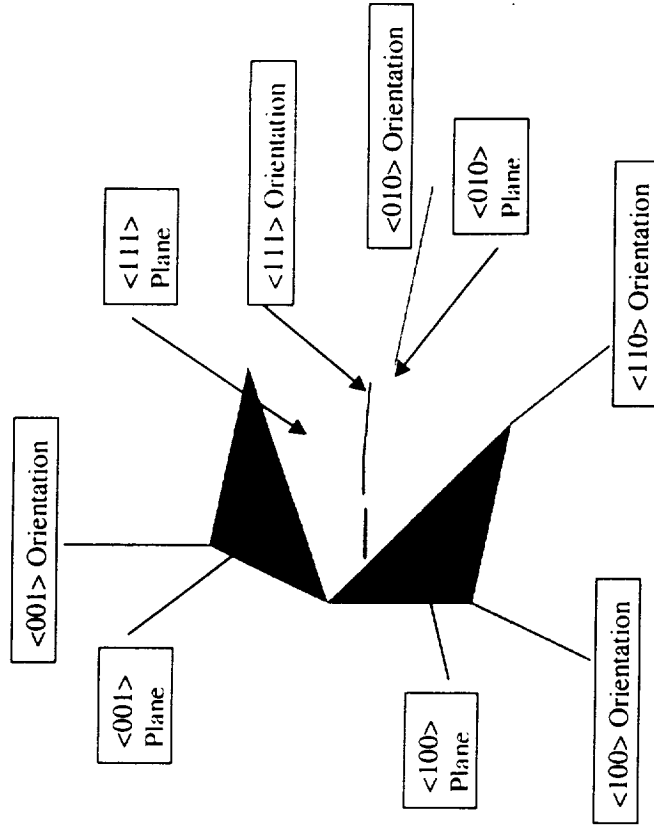


Fig. 2.3 FCC Crystallographic Orientation²



Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

Single Crystal Cracks

- Crack is on crystal $\langle 111 \rangle$ family of planes

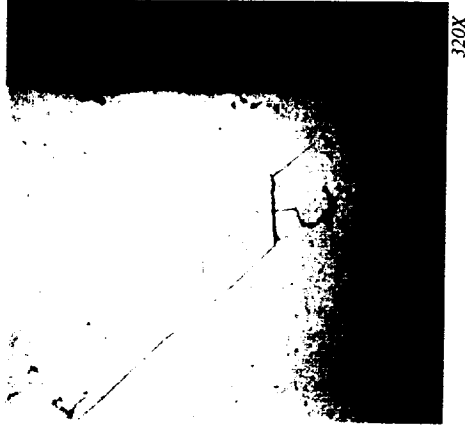


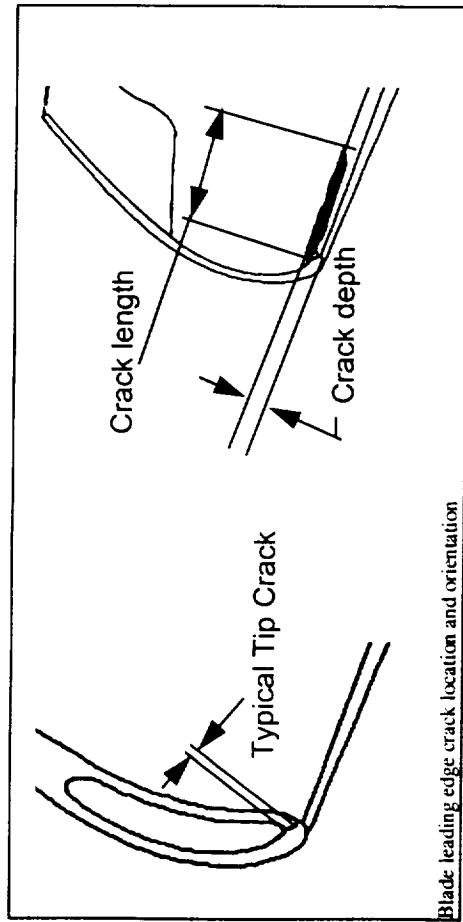
Fig. 3.3 A subsurface fretting fatigue crack emanating from a carbide in a turbine blade attachment (PWA1422) and propagating along octahedral $\{111\}$ shear planes.



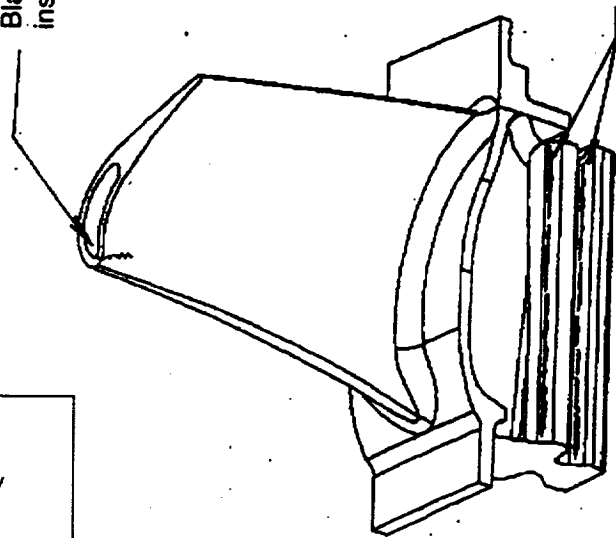
Fig. 3.4 Subsurface fretting fatigue crystallographic crack initiation in a single crystal Ni turbine blade platform⁵



Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Blade tip crack originating at
inside fillet radius



1st Turbine Blade

Blade attachment load face galling
cracks identified at edge of galling

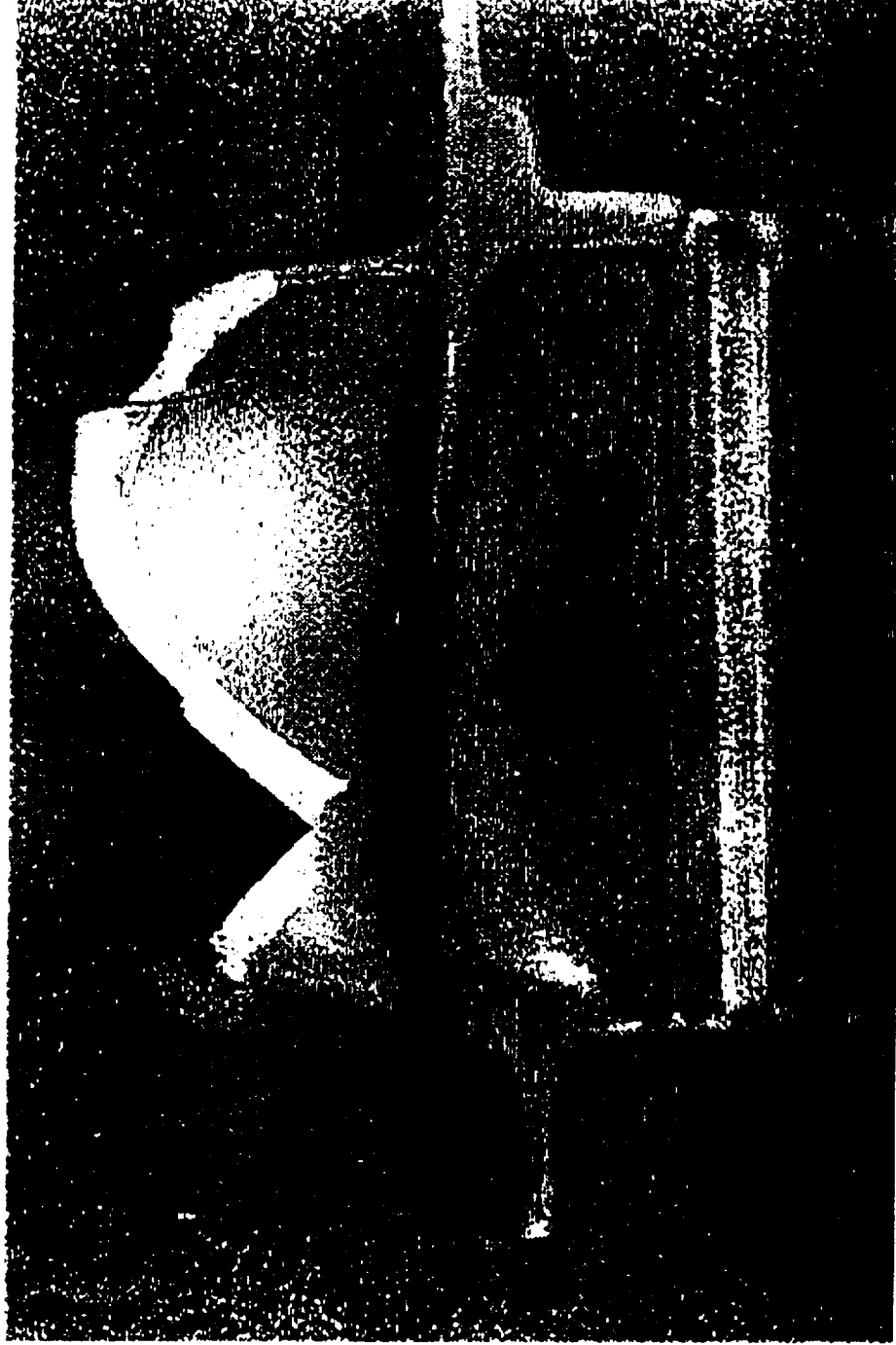
Development Blade Crack Locations

- Tip initiation at core leading edge (resolved for certification and flight by increasing core radius)
- Attachment at galling location (resolved for certification and flight by reducing attachment temperature, and by gold plating blade to reduce friction)

9/21/00



Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Development Blade Airfoil Failure

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Power Law Curve Fit ($R^2 = 0.468$): $\Delta\epsilon = 0.0238 N^{-0.124}$

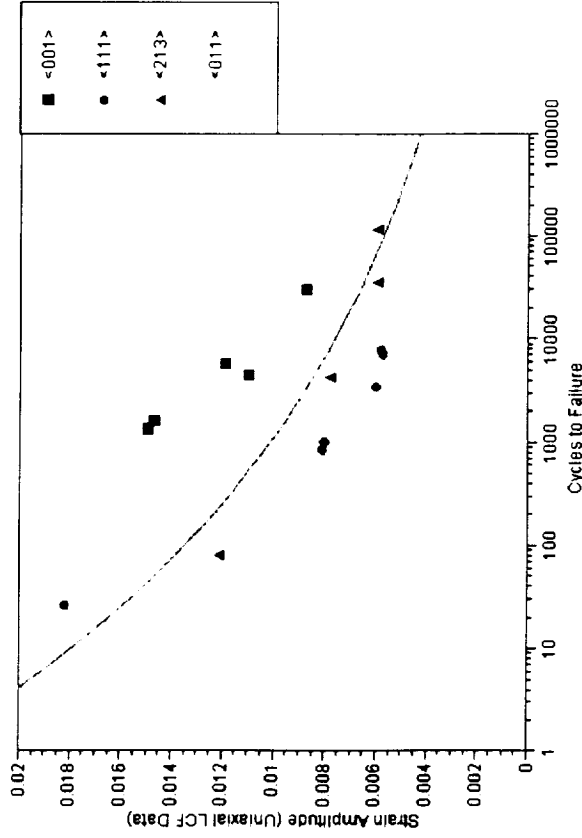


Fig. 4.3 Strain range Vs. Cycles to Failure for LCF test data (PWA 1484 at 1200F)

Power Law Curve Fit ($R^2 = 0.674$): $\Delta\tau = 397,758 N^{-0.1598}$

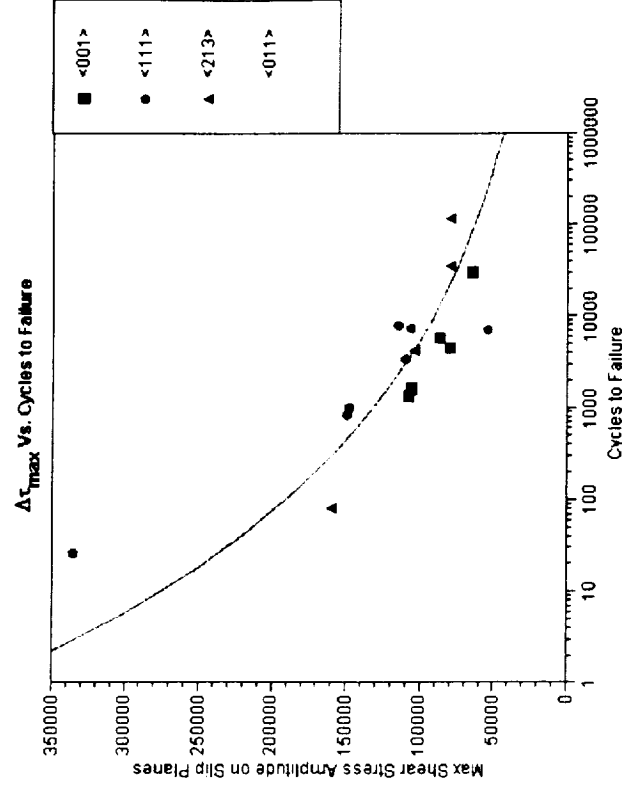


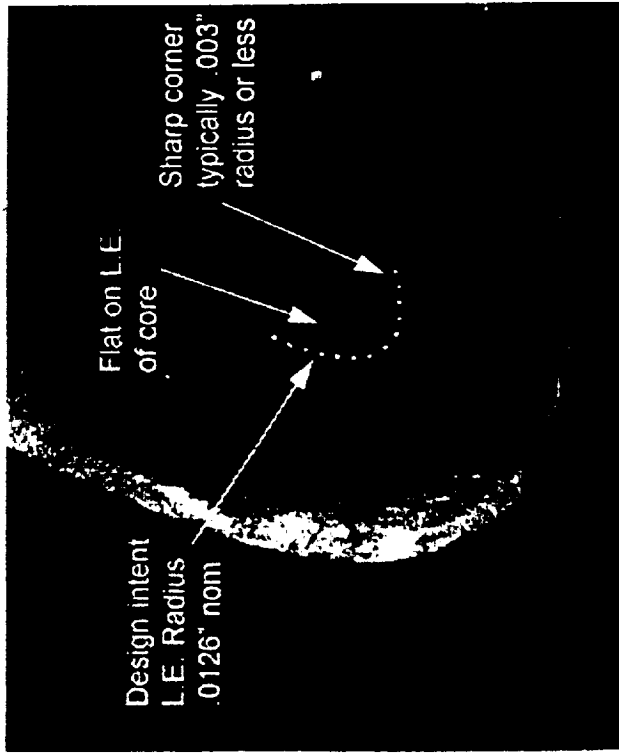
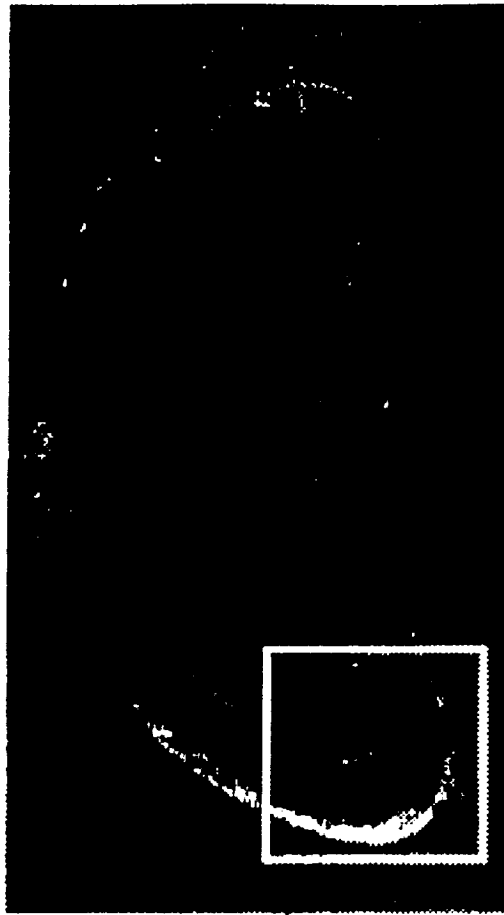
Fig. 4.8 Shear Stress Amplitude [$\Delta\tau_{\max}$] Vs. N

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Development Blade Core Profile

- Sharp reentrant corner
- Wall thickness variation
- Resolved for certification and flight by increasing core radius and tighter control of core placement



9/21/00

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Correlation Coefficient, R^2	Power Law Curve Fit
0.468	Strain Range, $\Delta\epsilon = 0.0238 N^{-0.124}$
0.130	$[\gamma_{\max} + \epsilon_n] = 0.0249 N^{-0.773}$
0.391	$\left[\frac{\Delta\gamma}{2} + \frac{\Delta\epsilon_n}{2} + \frac{\sigma_{no}}{E} \right] = 0.0206 N^{-0.101}$
0.383	$\left[\frac{\Delta\gamma}{2} (1 + k \frac{\sigma_{\max}}{\sigma_y}) \right] = 0.0342 N^{-0.143}$
0.189	$\left[\frac{\Delta\epsilon_1}{2} (\sigma_{\max}) \right] = 334.6 N^{-0.209}$
0.674	$\Delta\tau_{\max} = 397,758 N^{-0.1598}$ (Max shear stress amplitude of 30 slip systems)
0.744	$\Delta\tau_{\max} * \Delta\gamma/2 = 2,641 N^{-0.256}$
0.549	$\tau_{\max} * \Delta\gamma/2 = 4,661 N^{-0.227}$
0.775	$\Delta\sigma_{\text{vonMises}} = 845,607 N^{-0.157}$ [Equivalent stress (von Mises) amplitude]
0.775	$\Delta\tau_{\text{Tresca}} = 422,946 N^{-0.157}$ (Max principal shear stress amplitude)

Table 4.6 Power law curve fits for the failure parameters

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

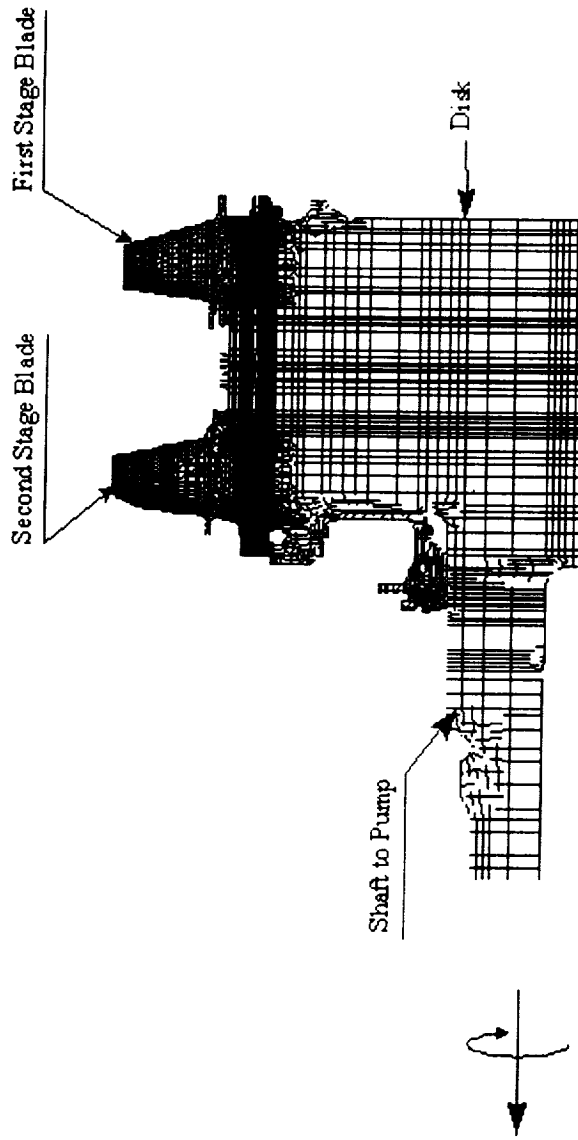


Figure 27. 3D ANSYS model of HPFTP/AT rotating turbine components.

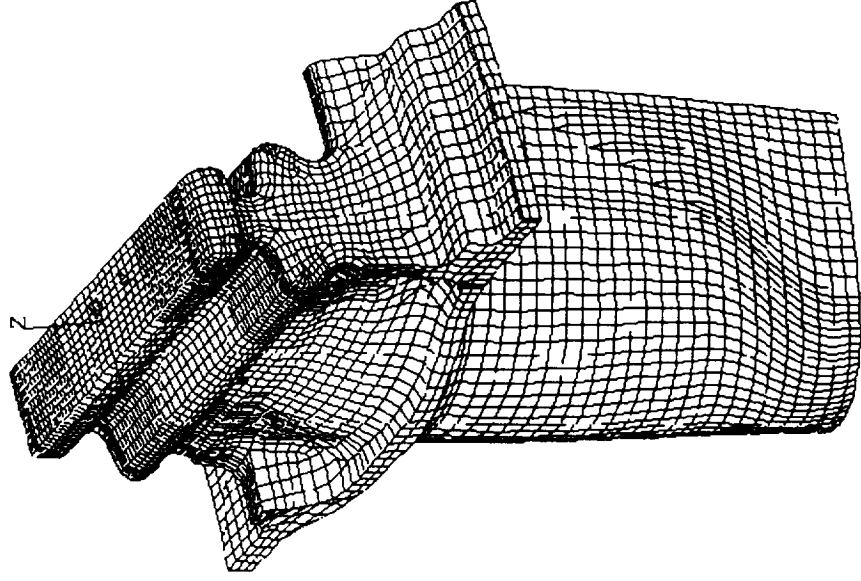


Figure 28. First Stage Blade Casting Coordinate System

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

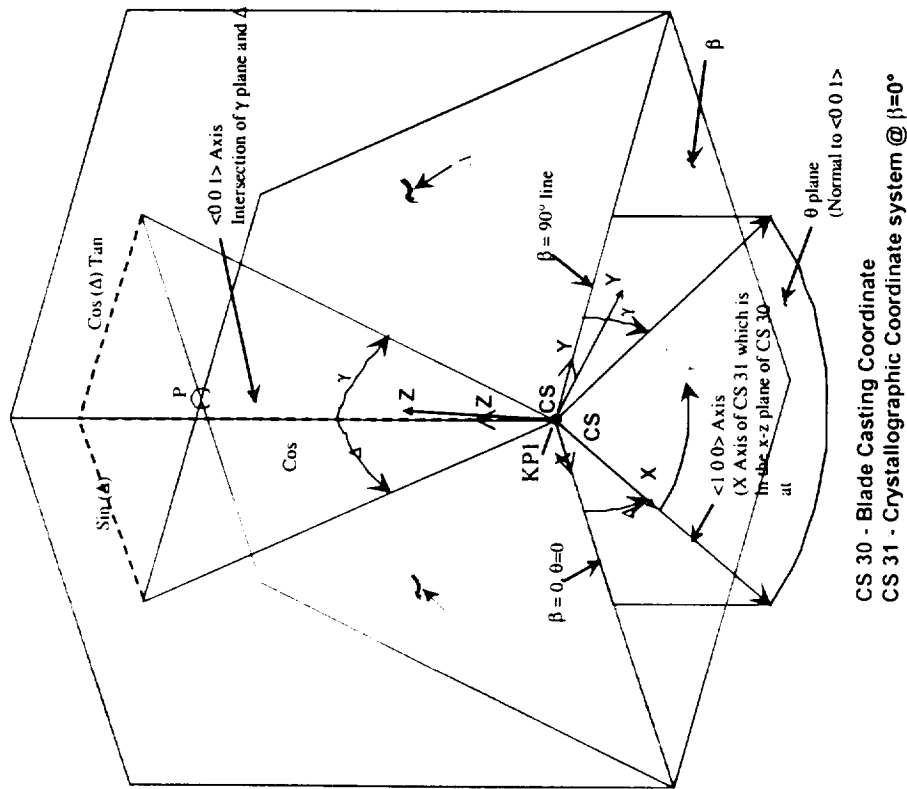
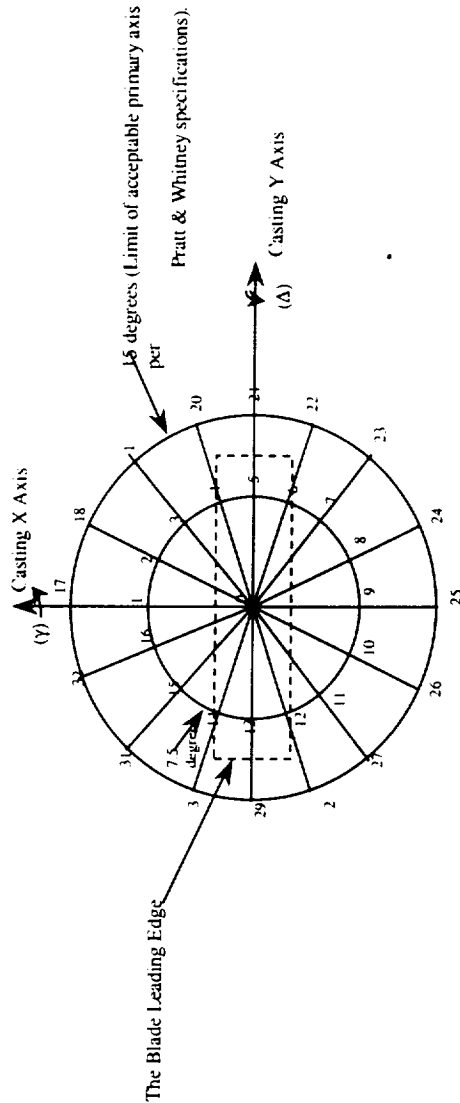


Figure 5.4. First stage blade material coordinate system relative to the casting coordinate system

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Top View of the Blade

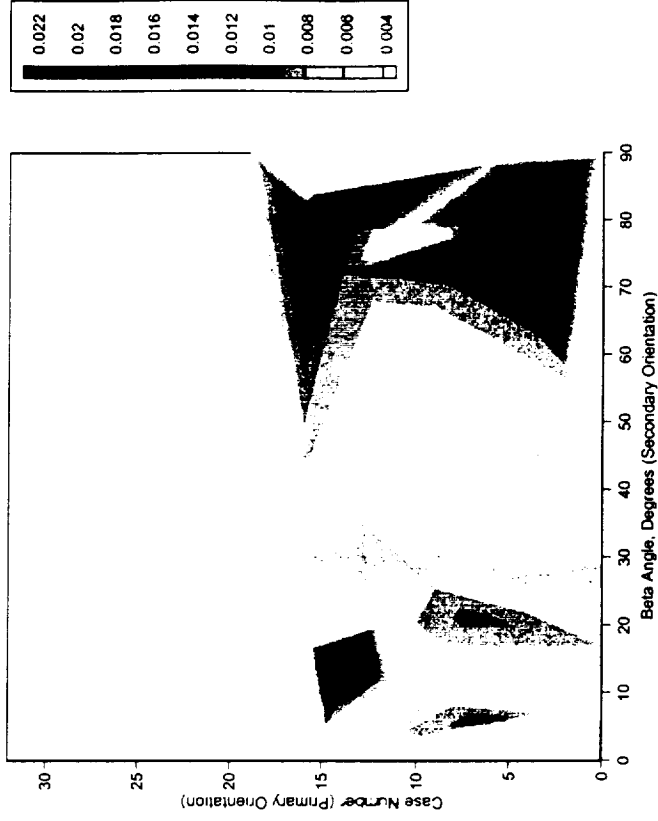
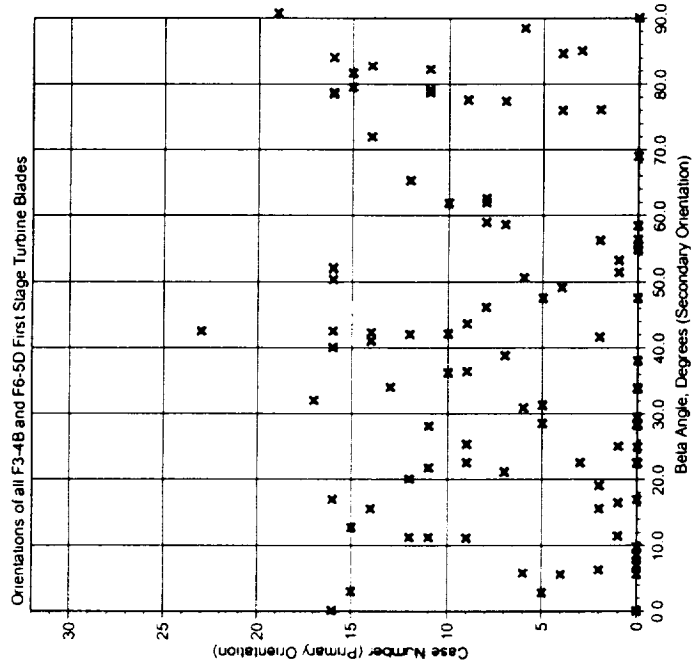
- * (33) orientations (cases) of (γ & Δ) with (9) (θ) orientations about each of these.
 - * 16 cases with (γ & Δ) combined to 7.5 degrees, 16 with (γ & Δ) combined to 15 degrees, 1 case with (γ & Δ) combined to 0 degrees.
- * (θ) varied from 0 to 80 degrees in 10 degree increments about the local material axis (due to symmetry 90 degrees is the same as 0 degree).
- * Total of (297) orientations.
- * Angles used in the model have not been reconciled with angles reported by casting vendor.

Case	Delta	Gamma	Beta
0	0.00	0.00	0.10 20.30 40.50 60.70 80
1	7.50	0.00	0.10 20.30 40.50 60.70 80
2	6.93	2.87	0.10 20.30 40.50 60.70 80
3	5.30	5.30	0.10 20.30 40.50 60.70 80
4	2.87	6.93	0.10 20.30 40.50 60.70 80
5	0.00	7.50	0.10 20.30 40.50 60.70 80
6	-2.87	6.93	0.10 20.30 40.50 60.70 80
7	-5.30	5.30	0.10 20.30 40.50 60.70 80
8	-6.93	2.87	0.10 20.30 40.50 60.70 80
9	-7.50	0.00	0.10 20.30 40.50 60.70 80
10	-6.93	-2.87	0.10 20.30 40.50 60.70 80
11	-5.30	-5.30	0.10 20.30 40.50 60.70 80
12	-2.87	-6.93	0.10 20.30 40.50 60.70 80
13	0.00	-7.50	0.10 20.30 40.50 60.70 80
14	2.87	-6.93	0.10 20.30 40.50 60.70 80
15	5.30	-5.30	0.10 20.30 40.50 60.70 80
16	6.93	-2.87	0.10 20.30 40.50 60.70 80
17	15.00	0.00	0.10 20.30 40.50 60.70 80
18	13.86	5.74	0.10 20.30 40.50 60.70 80
19	10.61	10.61	0.10 20.30 40.50 60.70 80
20	5.74	13.86	0.10 20.30 40.50 60.70 80
21	0.00	15.00	0.10 20.30 40.50 60.70 80
22	-5.74	13.86	0.10 20.30 40.50 60.70 80
23	-10.61	10.61	0.10 20.30 40.50 60.70 80
24	-13.86	5.74	0.10 20.30 40.50 60.70 80
25	-15.00	0.00	0.10 20.30 40.50 60.70 80
26	-13.86	-5.74	0.10 20.30 40.50 60.70 80
27	-10.61	-10.61	0.10 20.30 40.50 60.70 80
28	-5.74	-13.86	0.10 20.30 40.50 60.70 80
29	0.00	-15.00	0.10 20.30 40.50 60.70 80
30	5.74	-13.86	0.10 20.30 40.50 60.70 80
31	10.61	-10.61	0.10 20.30 40.50 60.70 80
32	13.86	-5.74	0.10 20.30 40.50 60.70 80

Table 5.1 33 primary axis cases with 9 secondary cases each, a total of 297 material orientations.

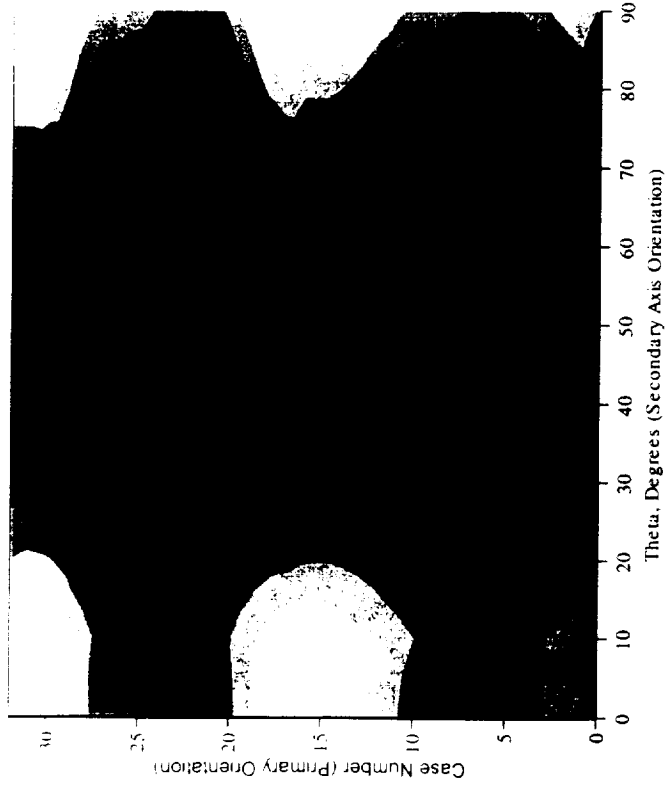
Figure 5.5 33 primary axis cases with 9 secondary cases each, a total of 297 material orientations.

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades

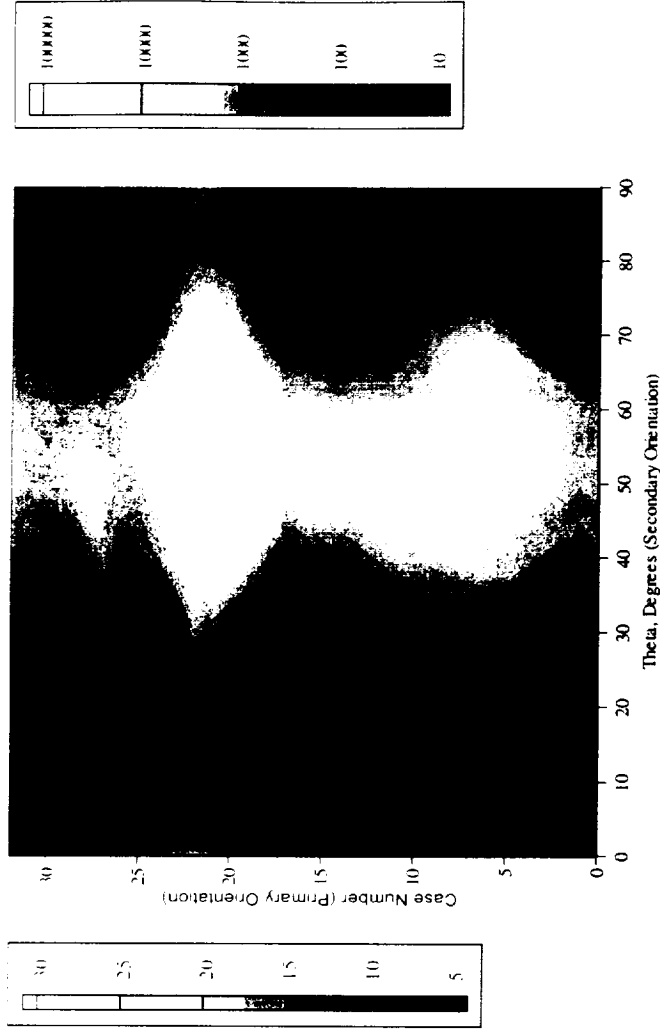


F3-4B and F6-5D First blades XXX FPI data, Contour Plot of
Crack Length Across Blade Wall in Inches (0.02 is through
wall for this plot)

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Maximum Resolved Shear Stress at Blade Tip



Dimensionless life at the first blade airfoil crack
initiation point.

Fatigue Failure of Development Space Shuttle Main Engine Turbine Blades



Final Comments

- HPFTP/AT is certified for flight
 - Development problems resolved before certification and flight
 - Final design has completed extensive hot fire testing
- Further Development of Analytical Tools for Single Crystal Materials
 - Currently in work
 - Needed as part of effort to further enhance blade and vane life
- Further Development of Crystallographic Crack Growth Specimens
 - Currently in work
 - Needed for determining the crack growth threshold for this material in the H₂ + Steam environment
- Reference
 - Swanson, G. R. and Arakere, N. K., "Effect of Crystal Orientation on Analysis of Single-Crystal, Nickel-Based Turbine Blade Superalloys", NASA/TP-2000-210074.